

# Hydrogen enriched compression ignition internal combustion engine: A dual fuel study

Experimental investigation

By

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Engine research group

Contributors:

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# Presentation Outline

1. Introduction
2. Experimental setup and cases
3. Results and analysis
4. Conclusion and future work

# Decarbonizing CI engines

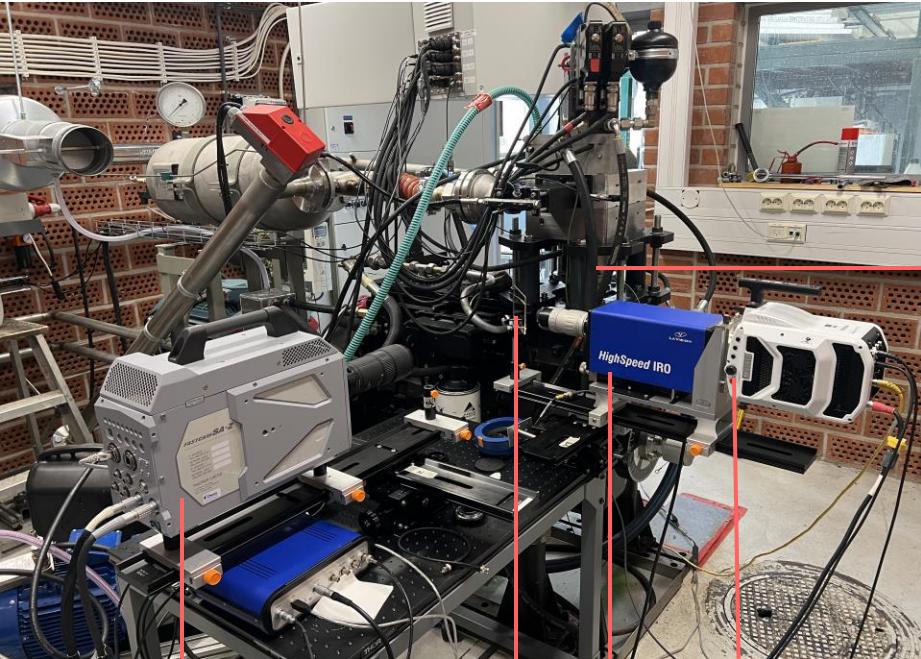


**Dual fuel compression ignition internal combustion engine**

# Research engines

## Optical engine

1. Single cylinder, 4-stroke engine.
2. Cylinder displacement =  $1.40 \text{ dm}^3$ .
3. Compression ratio  $\sim 14.2:1$ .
4. Electrohydraulic valve actuation system (EHVA).
5. Optically accessible.
6. Rpm  $\sim 800\text{-}1200$ .

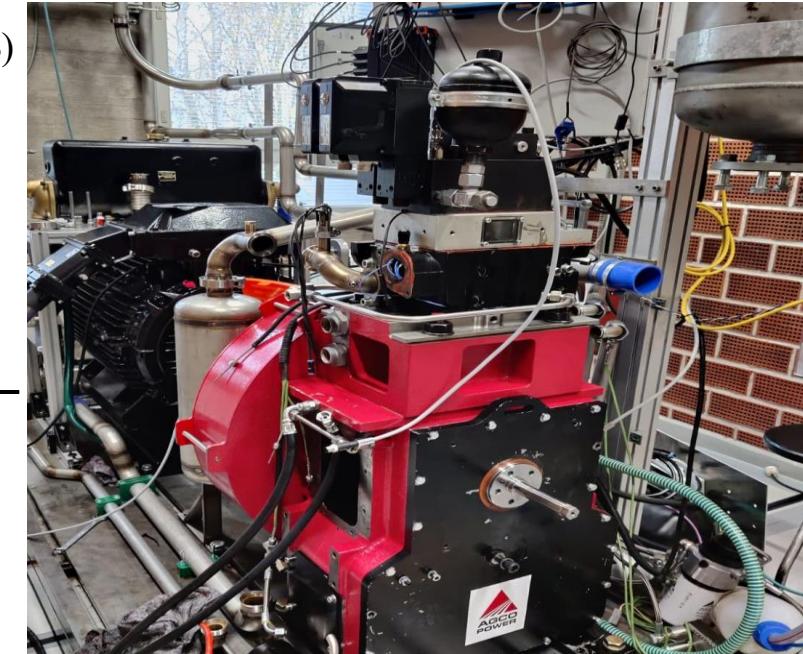
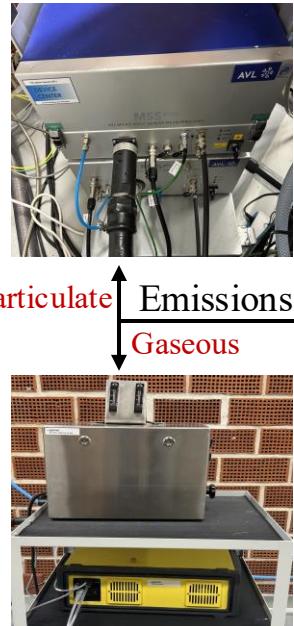


High speed color camera  
Beam splitter  
High speed intensifier  
High speed monochrome camera  
Optical piston assembly

## Full metal engine

1. Single cylinder, 4-stroke engine.
2. Cylinder displacement =  $1.25 \text{ dm}^3$ .
3. Compression ratio  $\sim 17.3:1$ .
4. Electrohydraulic valve actuation system (EHVA).
5. Full metal cylinder engine.
6. Rpm  $\sim 800\text{-}2000$ .

Micro-soot sensor (MSS)



Fourier transform infrared (FTIR)

# Full metal engine test cases

Parameter	Value
RPM	1000
Intake air mass flow rate (kg/h)	40
Hydrogen energy share (in %)	0,10,20,30,40,50
Global lambda	1.47-1.59
Gas lambda	17.49-3.50
CA50 (CA Degrees)	6,8,10
Diesel injection pressure (bar)	1000
Total input fuel energy (MJ/h)	80

- Direct injection (non-premixed) strategy for introducing diesel.
- Port fuel injection (premixed) strategy for introducing hydrogen.

\* Please note CA50 → Crank angle at which 50% of total heat release has been achieved.

# Optical engine test cases

Parameter	Value
RPM	1000
Intake air mass flow rate (kg/h)	30
Total input fuel energy (MJ/h)	60
Diesel injection pressure (bar)	1200
Hydrogen injection strategy	PFI
Start of diesel injection (DBTDC)	8
Hydrogen energy share (in %)	0,10,20,30,40
Gas lambda	17.65-4.41

## 30% H<sub>2</sub> energy share

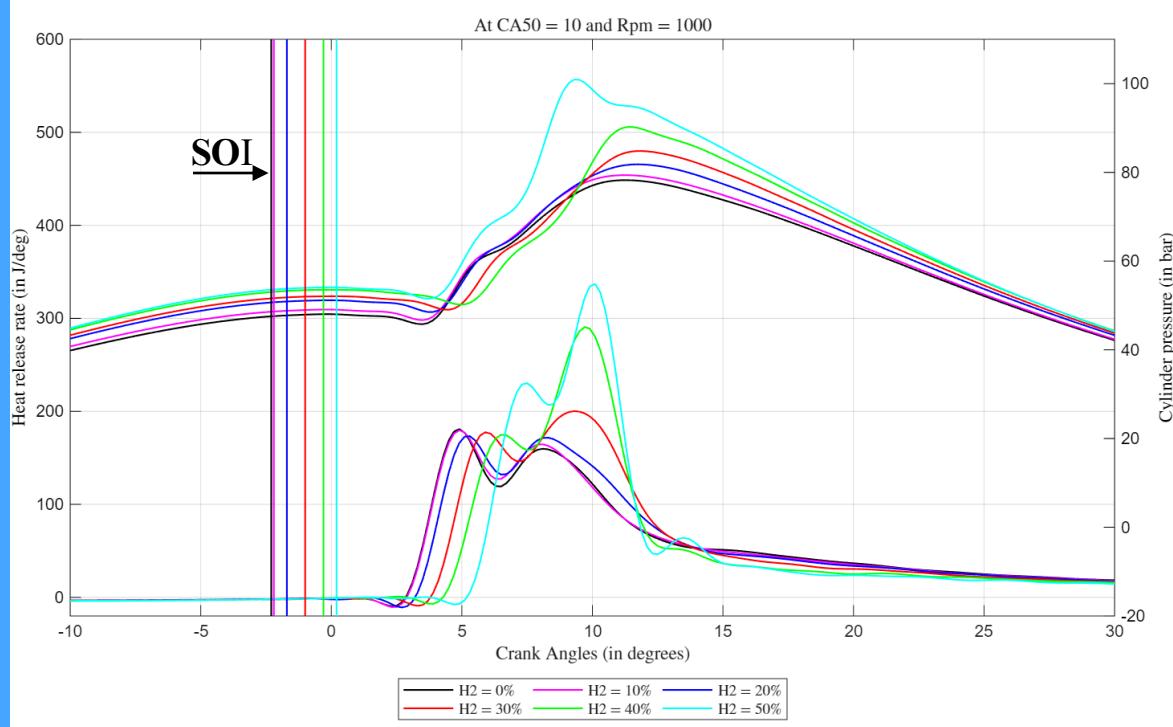
Parameter	Value
Start of diesel injection (DBTDC)	8,10,12

- Skip firing strategy (firing every 7<sup>th</sup> cycle).

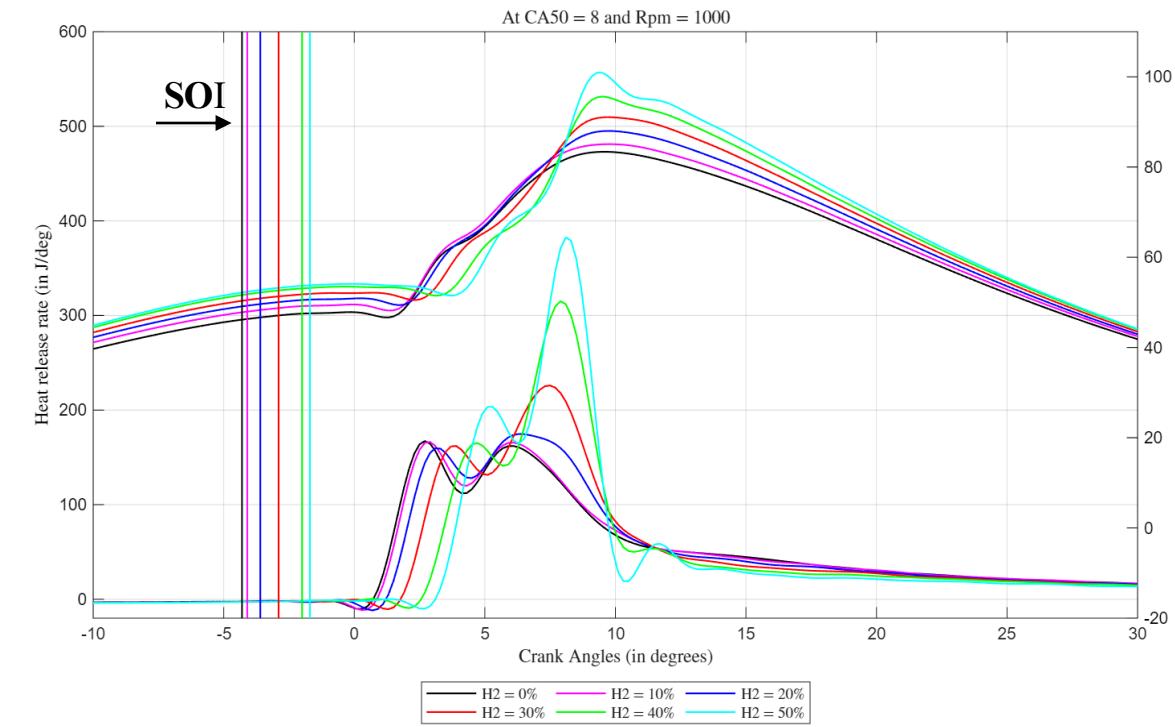
# FULL METAL ENGINE RESULTS

## COMBUSTION ANALYSIS

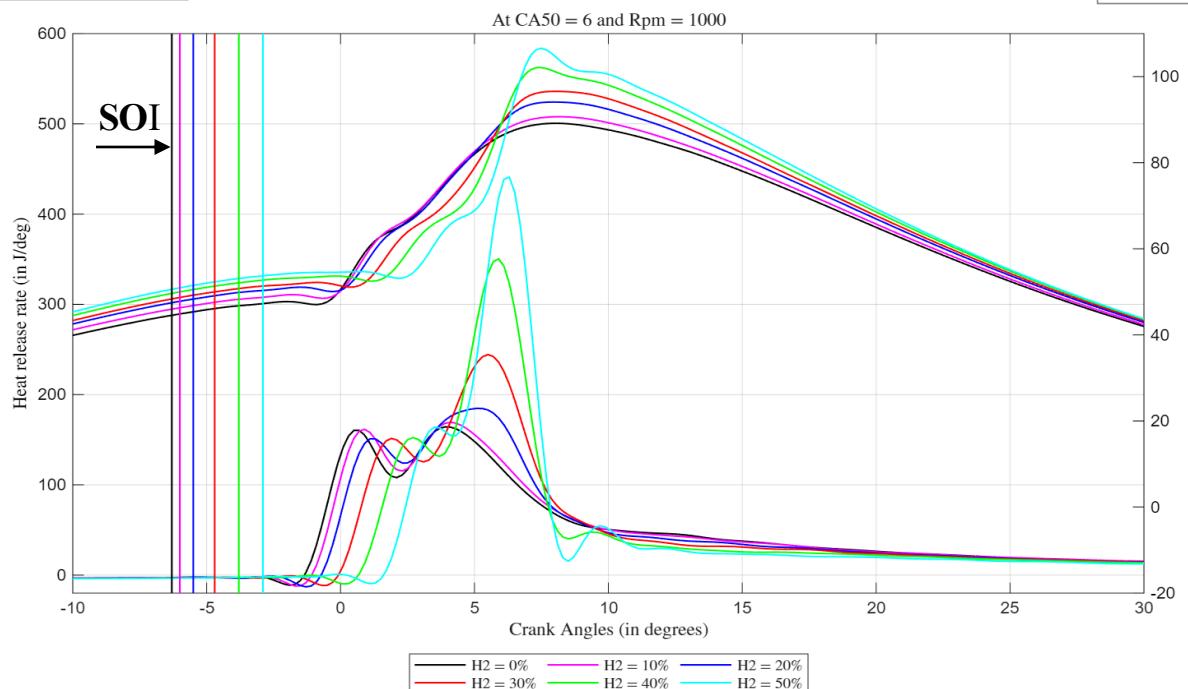
# Experimental results



- First HRR peak results from diesel.
- Second HRR peak results from premixed hydrogen.
- \* Please note Start of injection (SOI) is in CA Degrees.



- With an increase in H2 energy share the peak cylinder pressure and HRR increases for all CA50.
- With early CA50 the second HRR peak (related to premixed H2) increases.

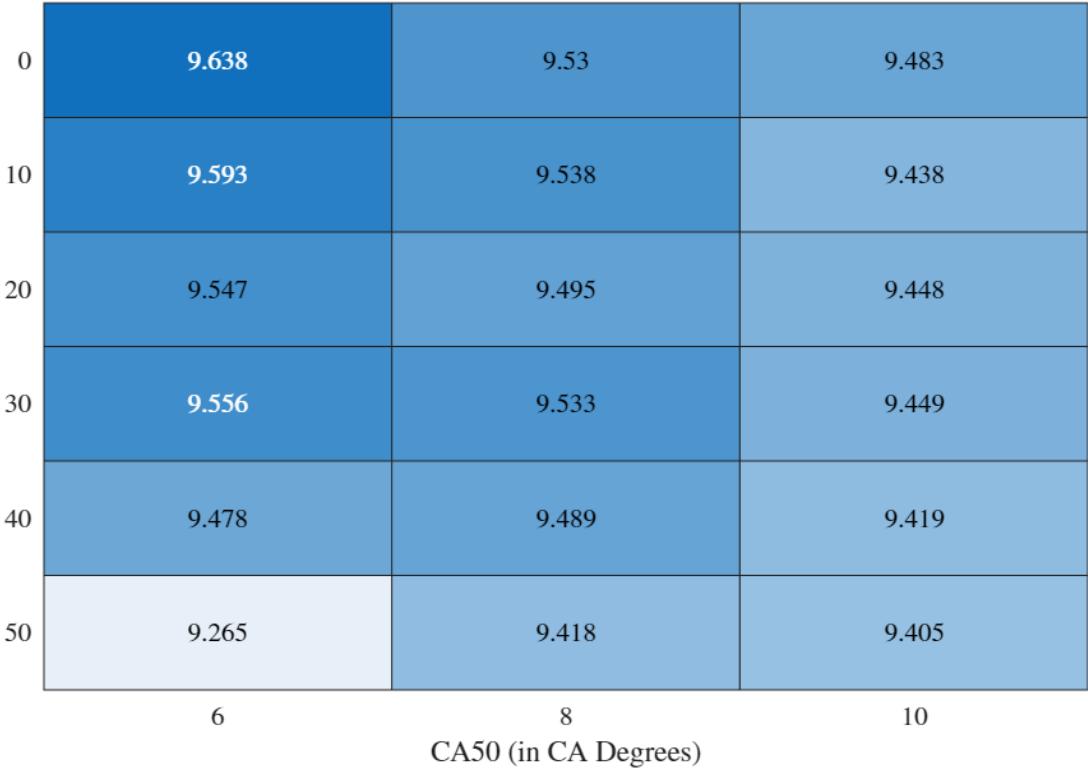


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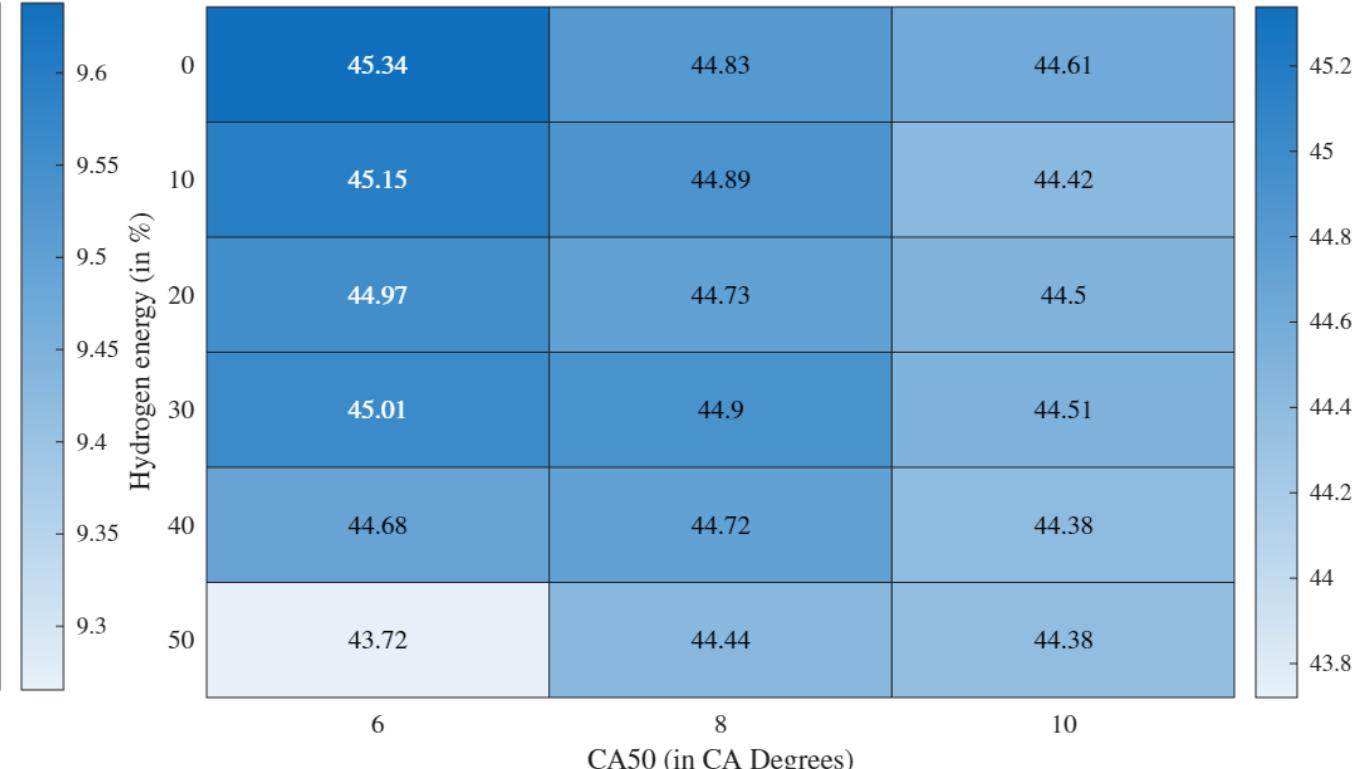
# PERFORMANCE METRICS

# Experimental results

Indicated mean effective pressure at 1000 rpm (in bar)



Indicated thermal efficiency at 1000 rpm (in %)



➤ IMEP and ITE remains nearly constant.

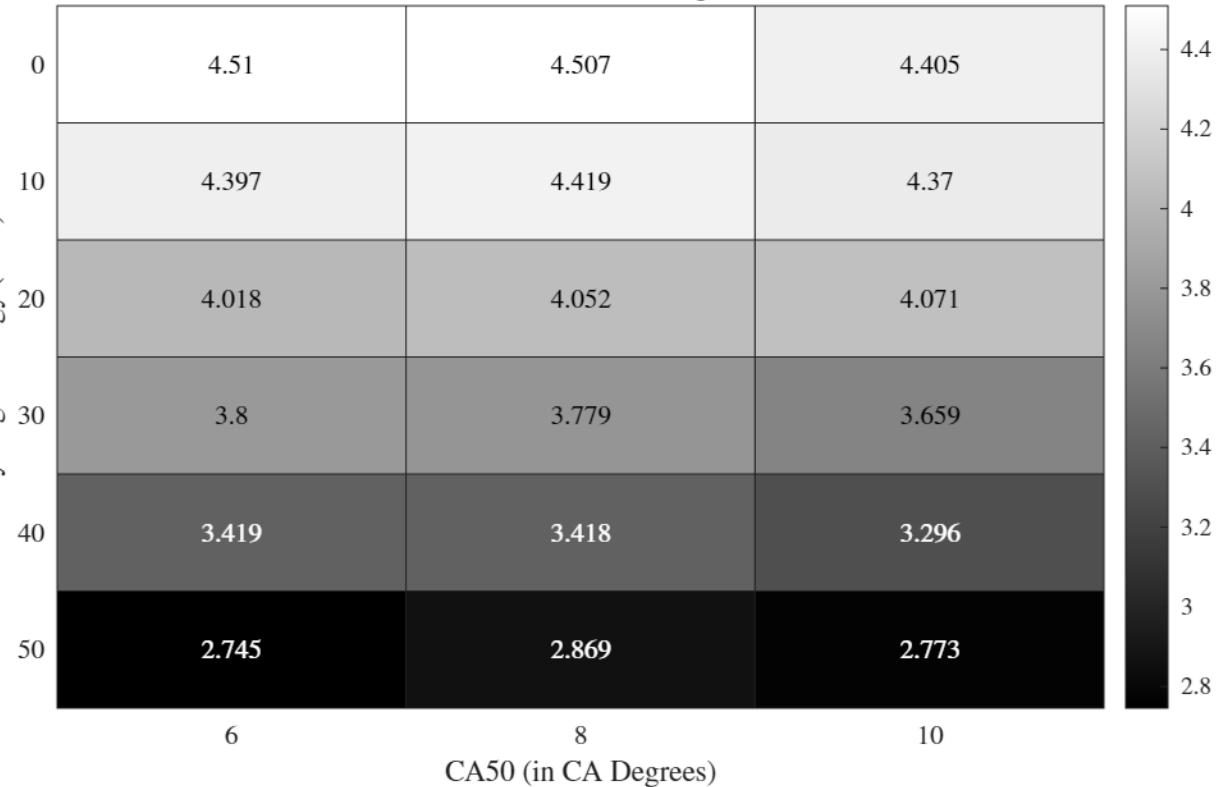


Highlights the constant total energy condition.

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# Experimental results

Combustion duration at 1000 rpm (in ms)

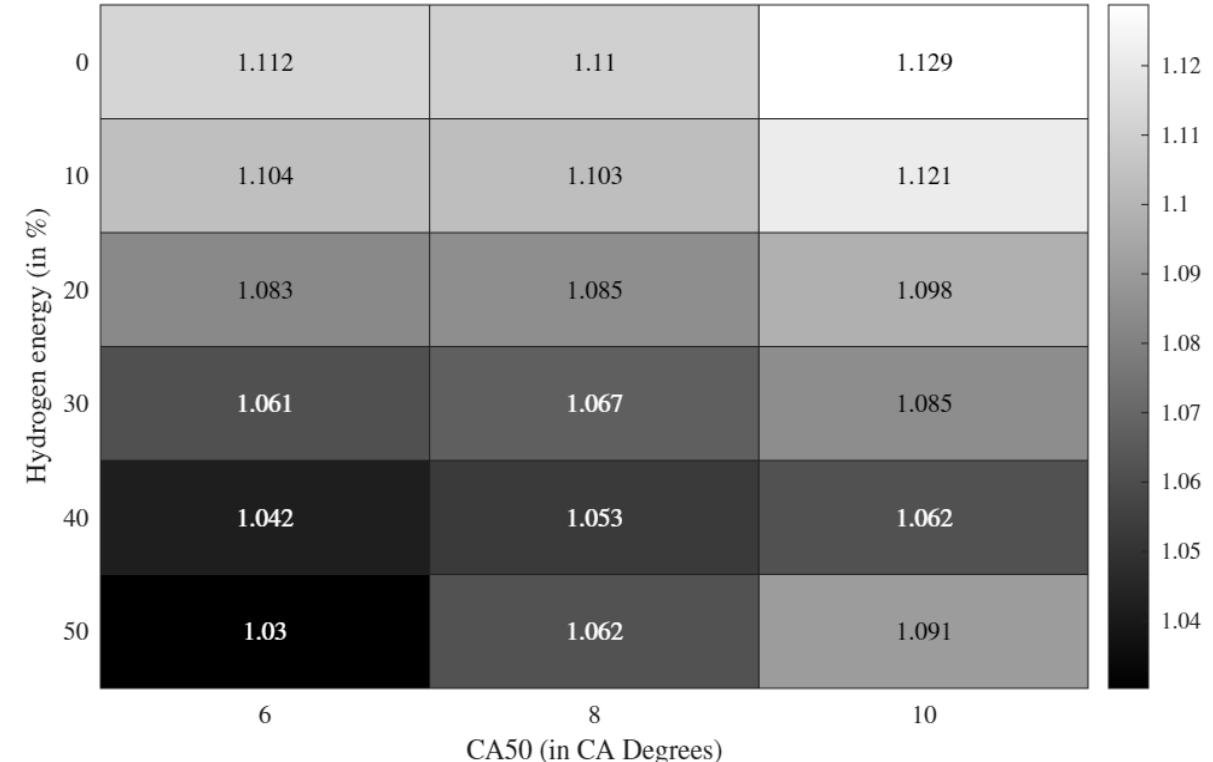


➤ CD decreases with increasing HES



Indicating faster reaction rates.

Ignition delay at 1000 rpm (in ms)



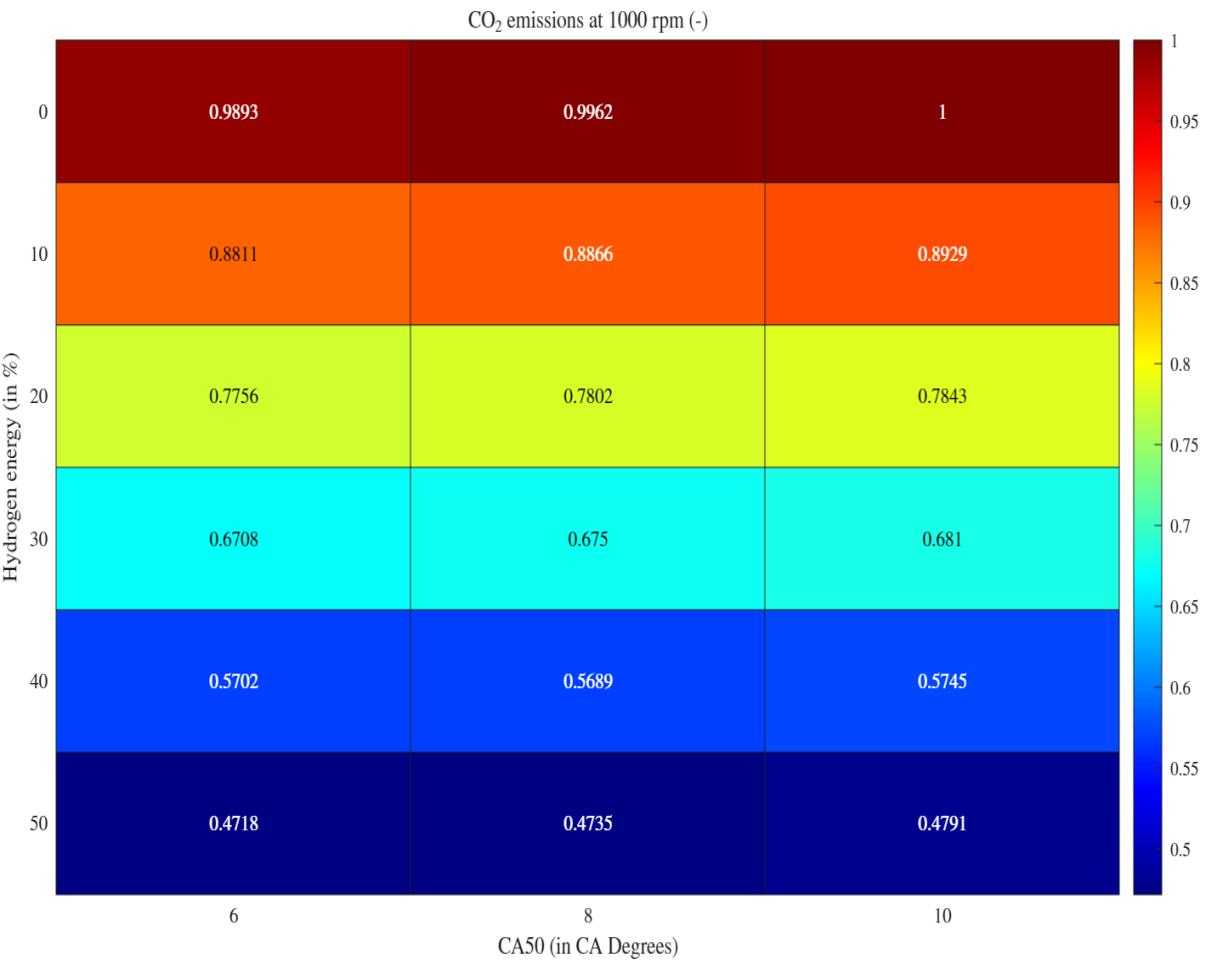
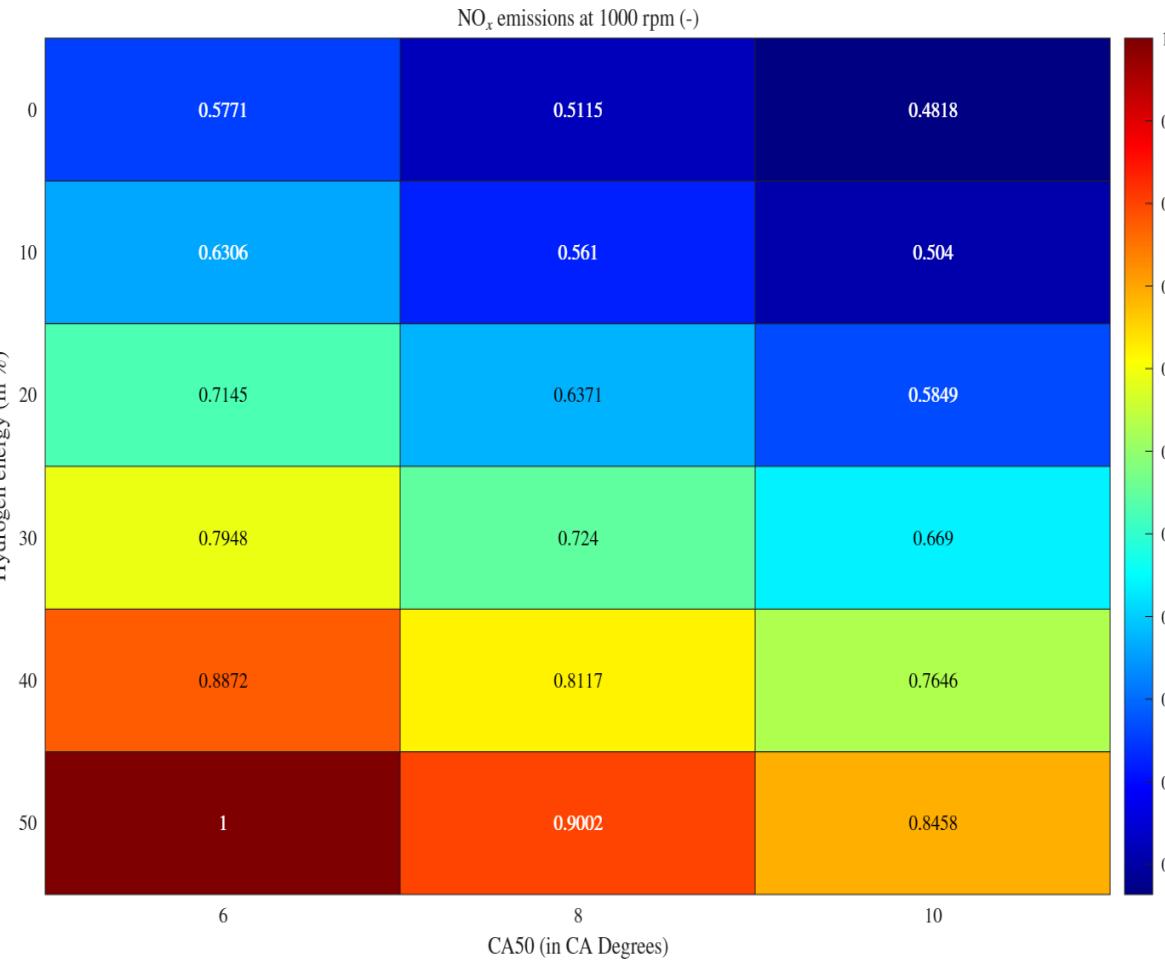
➤ IDT remains constant for all the cases.

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\* Please note CD = CA90 – CA05, IDT = CA05 – SOI.

# EMISSIONS

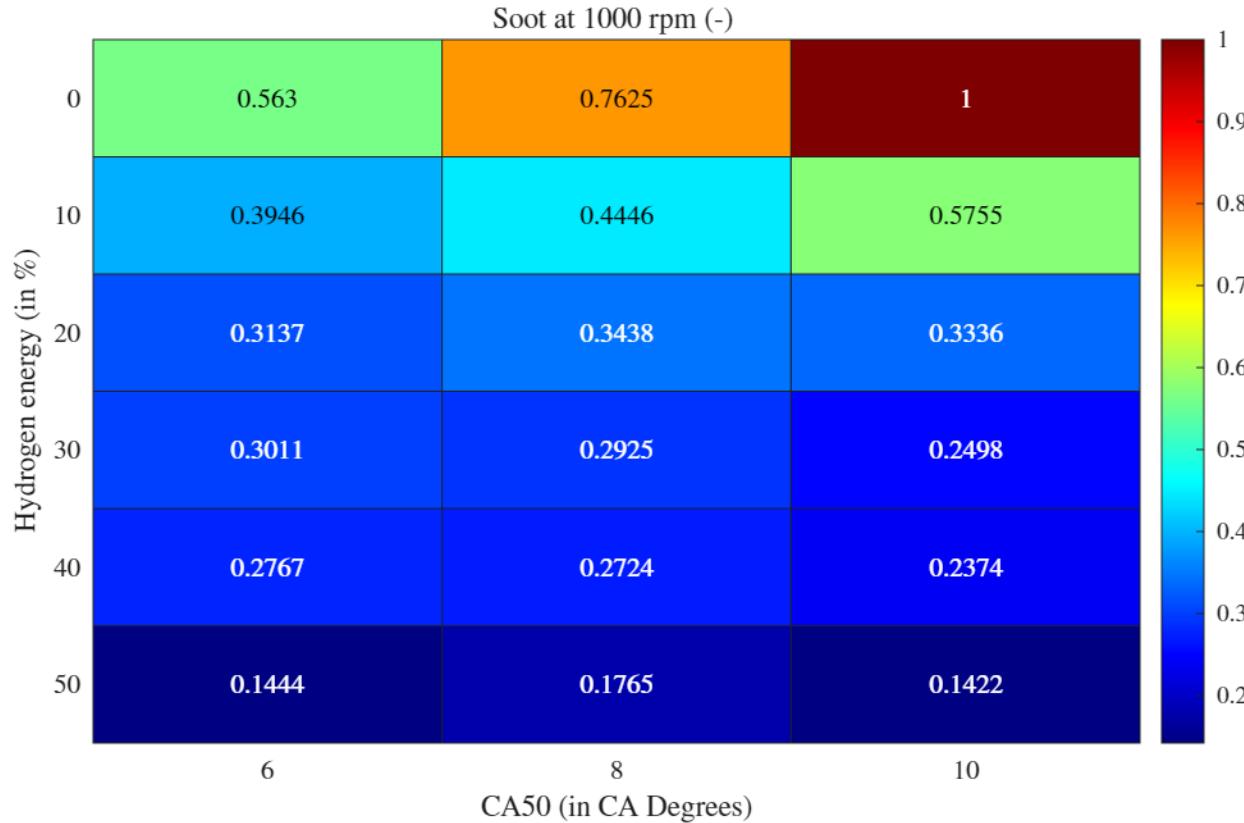
# Experimental results



- At any constant CA50 with increase in H2 energy share NOx increases.
- At constant H2 energy share NOx decreases with later CA50 values.

- At any constant CA50 with increase in H2 energy share CO2 decreases.
- CO2 emissions are independent of CA50 values.

# Experimental results



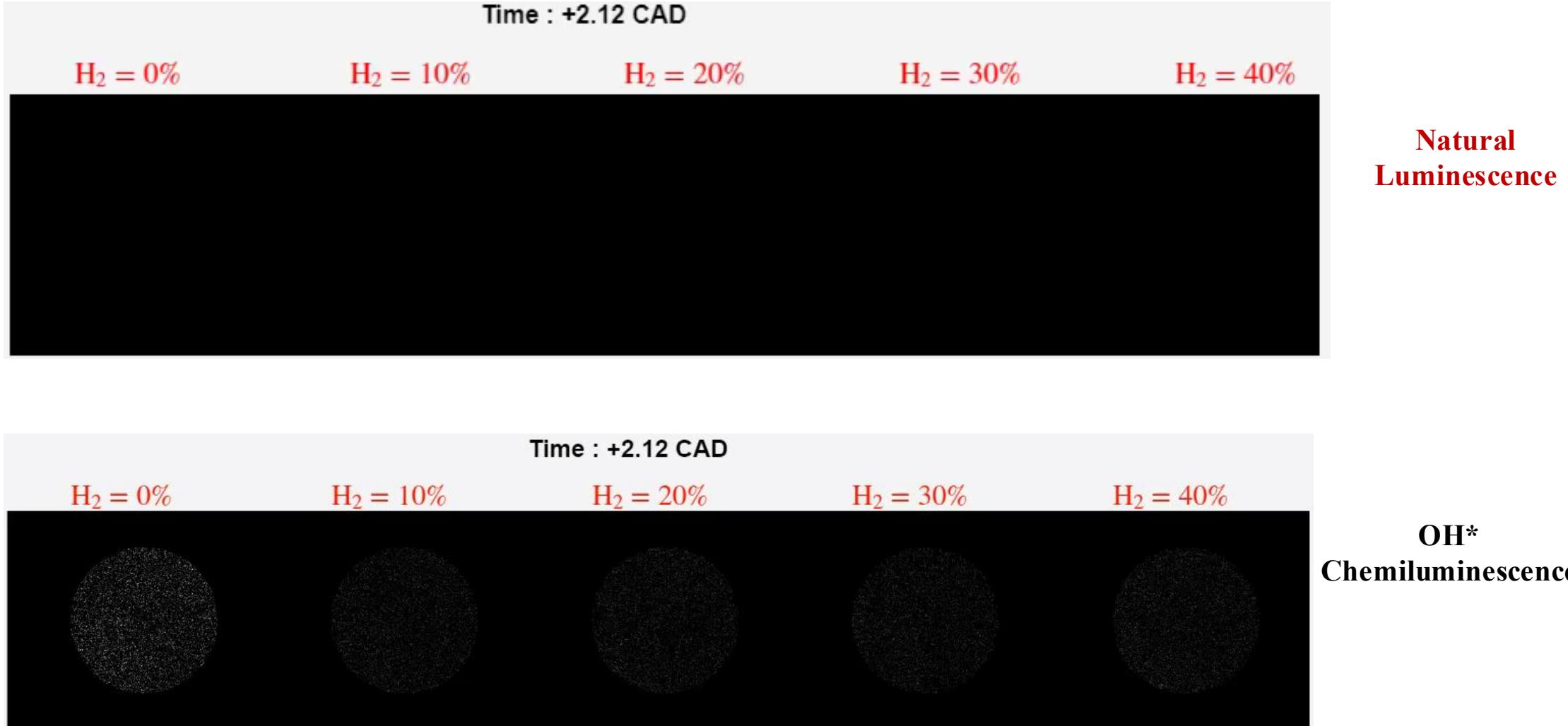
- At any constant CA50 with increase in H<sub>2</sub> energy share soot emissions decrease.
- Soot emissions do not seem to be dependent upon combustion phasing.

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# OPTICAL ENGINE RESULTS

# Experimental results

## H<sub>2</sub> energy ratio swept between 0 - 40% at constant SOI of 8 DBTDC



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➤ H<sub>2</sub> increases → Later ignition, Natural luminescence decreases, OH\* increases

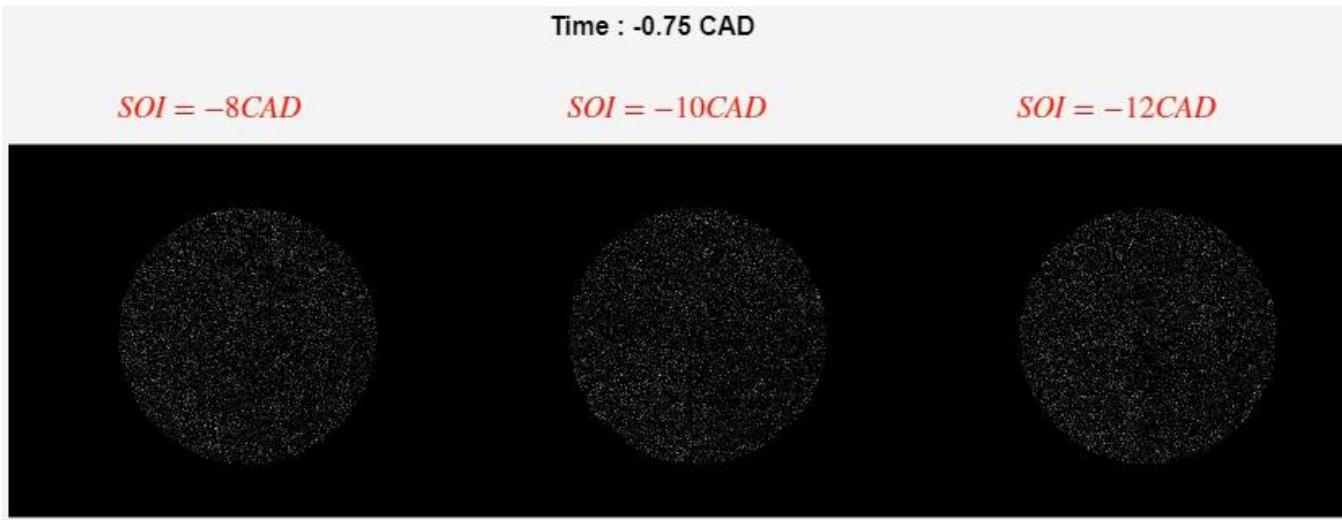
Decreased soot can be observed with decreasing flame natural luminescence intensity.

# Experimental results

SOI swept between 8-12 DBTDC at constant H2 energy ratio of 30%



**Natural Luminescence**



**OH\* Chemiluminescence**

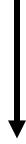
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➤ Early injection → Early ignition

# Summary & future work

Parameters	Hydrogen fraction	CA50
Heat release rate ↑	↑	Early ↑
NOx emissions ↑	↑	Early ↑
CO2 emissions ↑	↓	Independent
Soot emissions ↑	↓	Independent

- IDT in optical engine tests > full metal engine with increase in HES.



Check using chemical kinetics simulations.

- Limitations with HES (knocking observed)



Playing with lambda values (Full metal engine + Optical assessment).

# Acknowledgement

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**BUSINESS  
FINLAND**



**Flexible Clean  
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Technologies**



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# THANK YOU